

Gears with top in-service performance developed for hybrid and electric vehicles – TOPGEAR

Workshop in Aachen 08. April, 2025

Laboratory for Machine Tools and Production Engineering (WZL)
RWTH Aachen University
Chair of Machine Tools



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Gear Manufacturing

Gear Manufacturing Process Chain of both Material Variants

Variant 01: Reference Material – 27MnCr5



Pitting Test Gear
for high speed tests



Scuffing Test Gear
for high speed tests

Variant 02: 40CrMoBi4



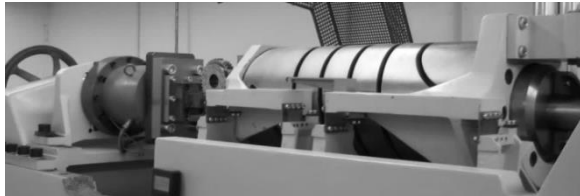
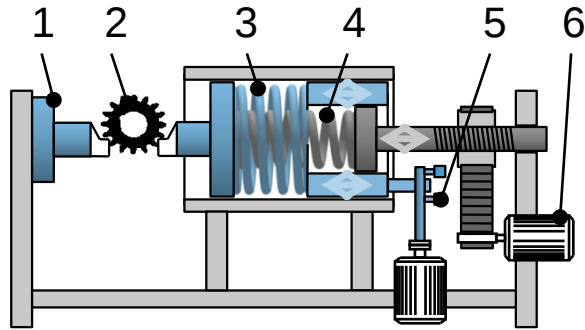
Agenda

1	Tooth Root Bending Strength
2	Flank Load Capacity
3	Scuffing Load Capacity
4	Conclusions

Operating Principles of Pulsator Test Rigs

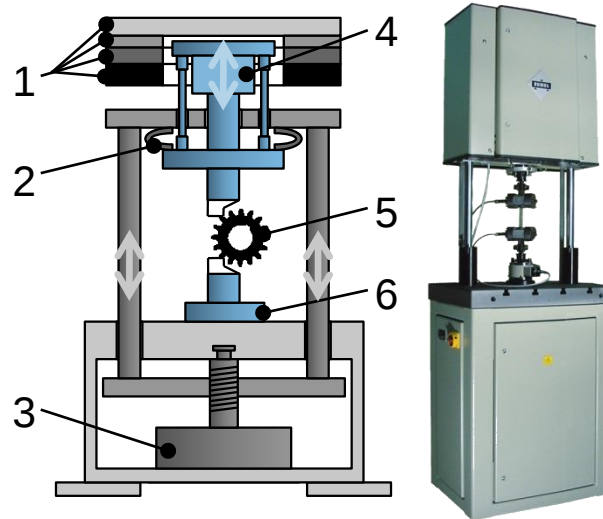


Imbalance-Excited Resonance Pulsator



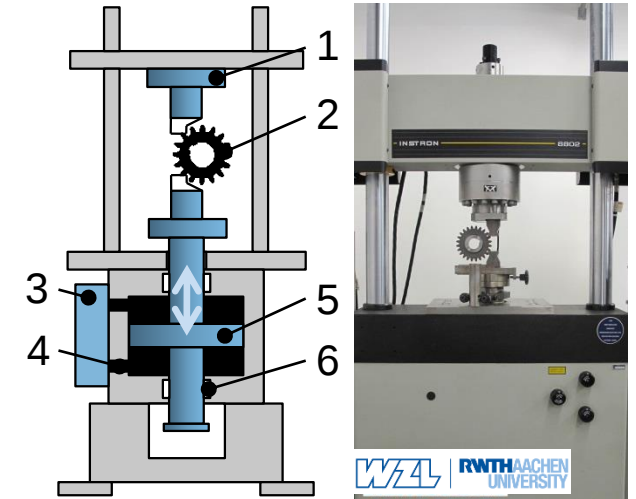
- 1 Load Cell
- 2 Test Gear
- 3 Pulsating Spring
- 4 Medium Load Spring
- 5 Centrifugal Exciter
- 6 Medium Load Drive

Magnetic Resonance Pulsator



- 1 Variable Excitation Mass
- 2 Medium Load Spring
- 3 Medium Load Drive
- 4 Excitation Magnet
- 5 Test Gear
- 6 Load Cell

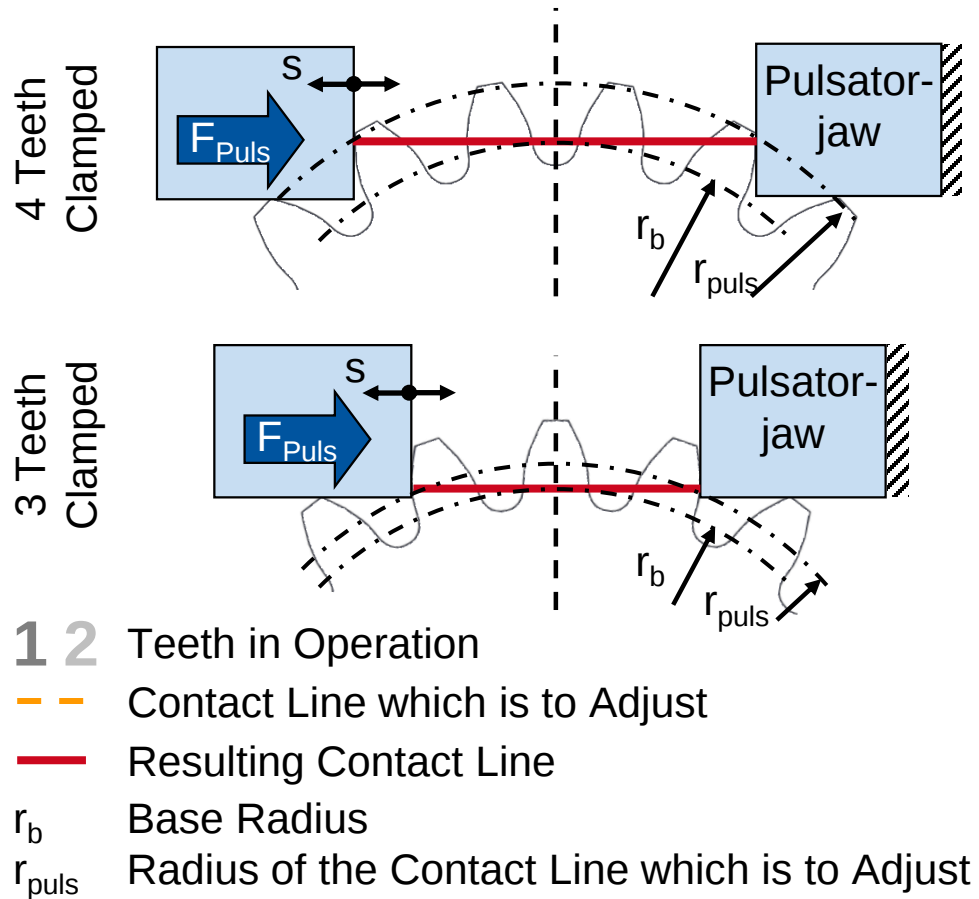
Hydraulic Pulsator



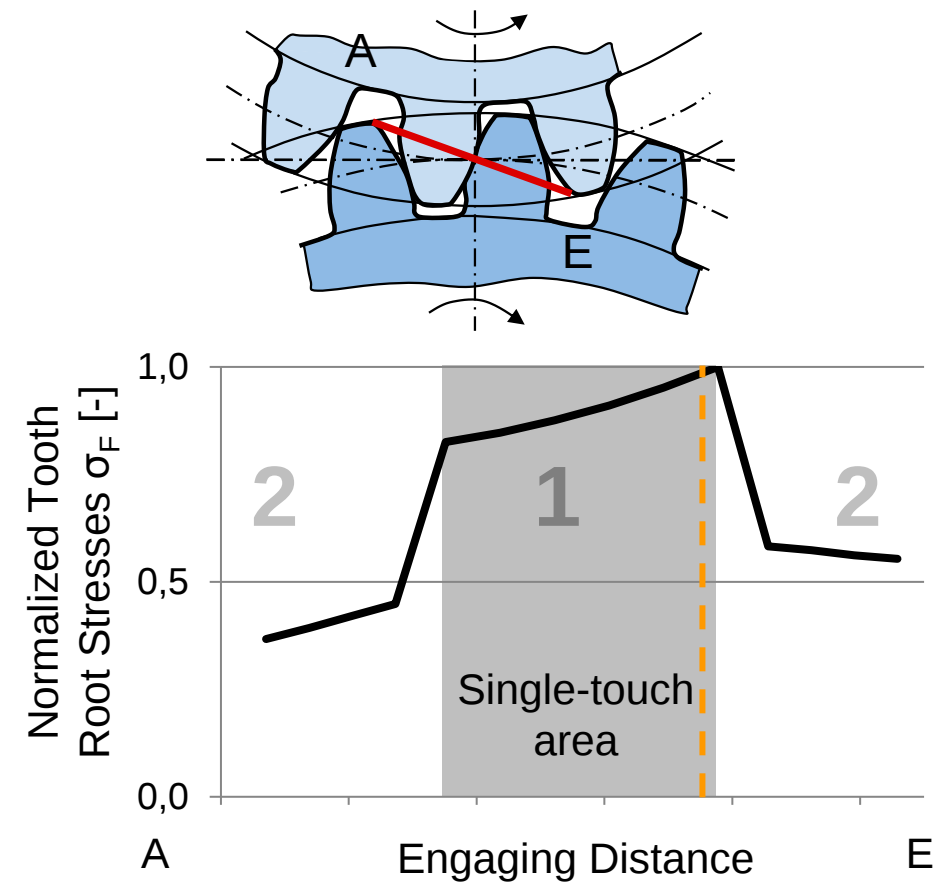
- 1 Load Cell
- 2 Test Gear
- 3 Servo Valve
- 4 Oil Input
- 5 Piston
- 6 Hydrostatic Bearing

Comparison of Stresses of Running Test and Pulsator Test

Location of the Contact Line Depends on the Number of Teeth

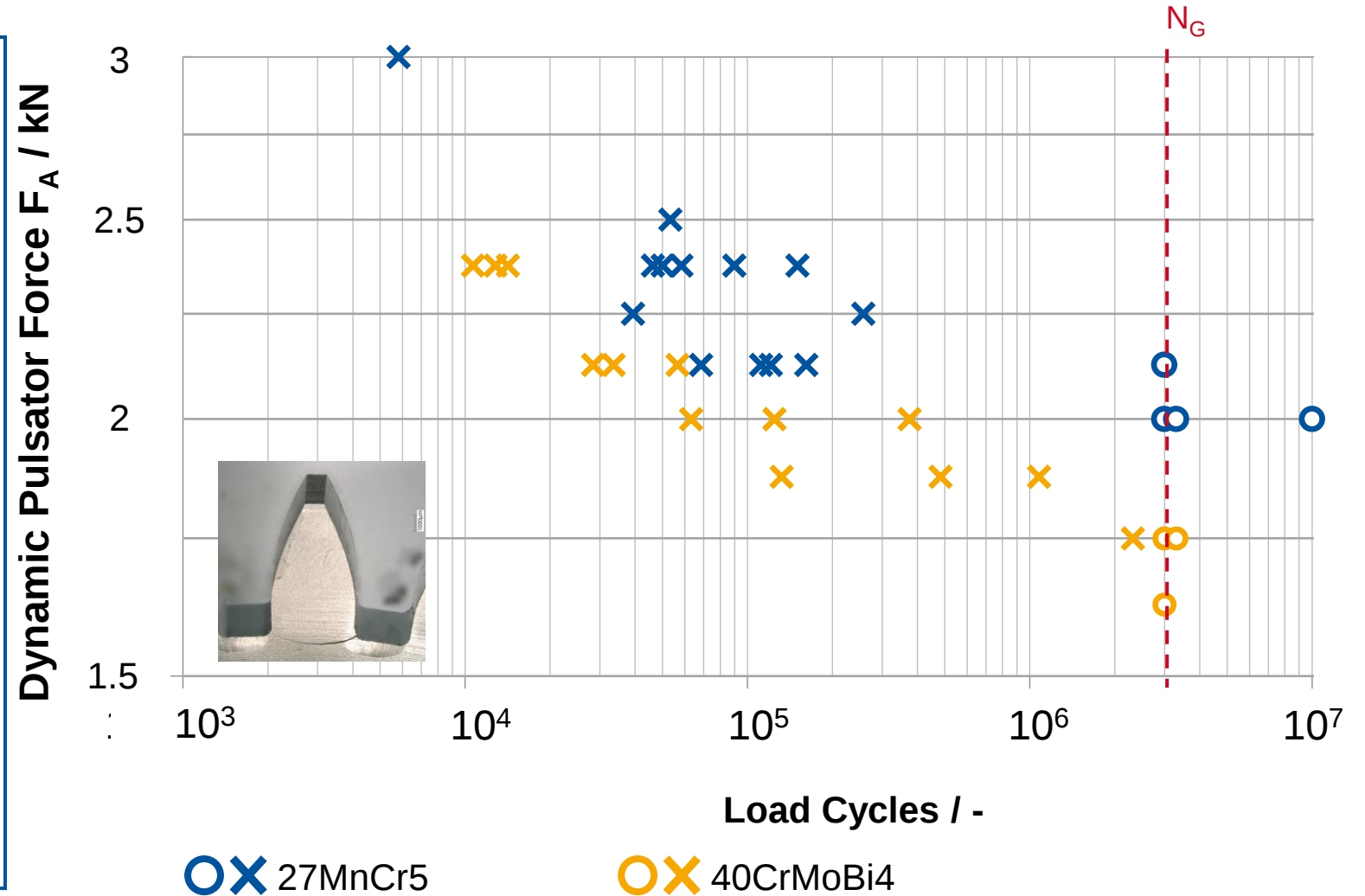
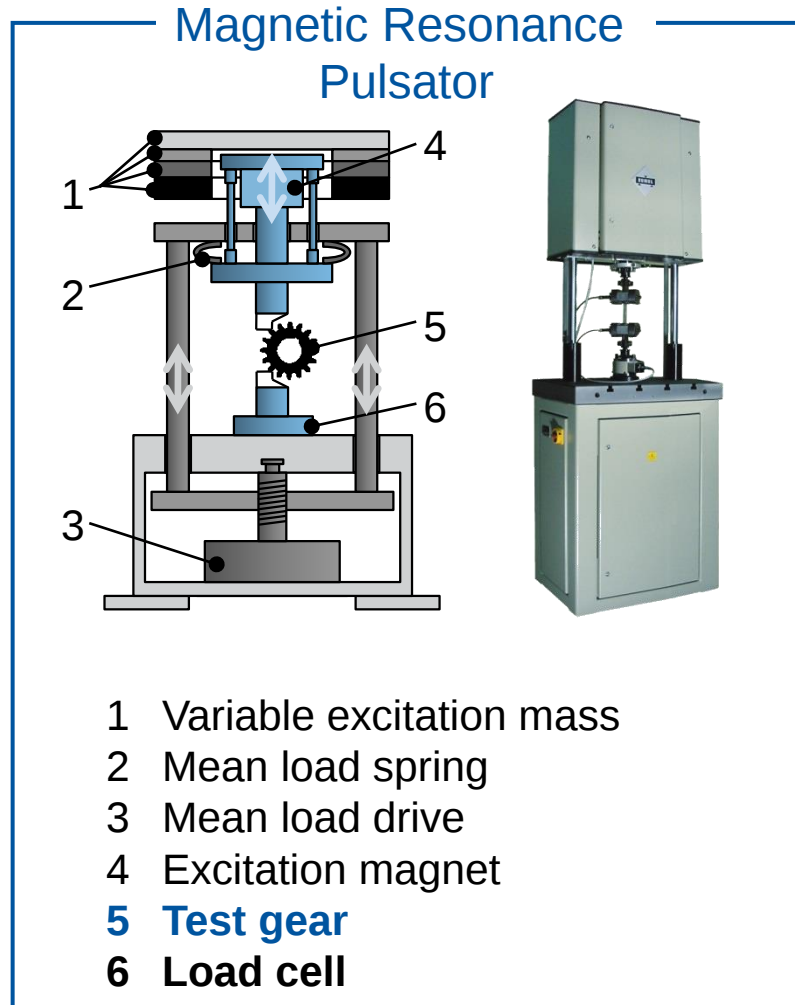


Pulse line next to the outer single-touch area



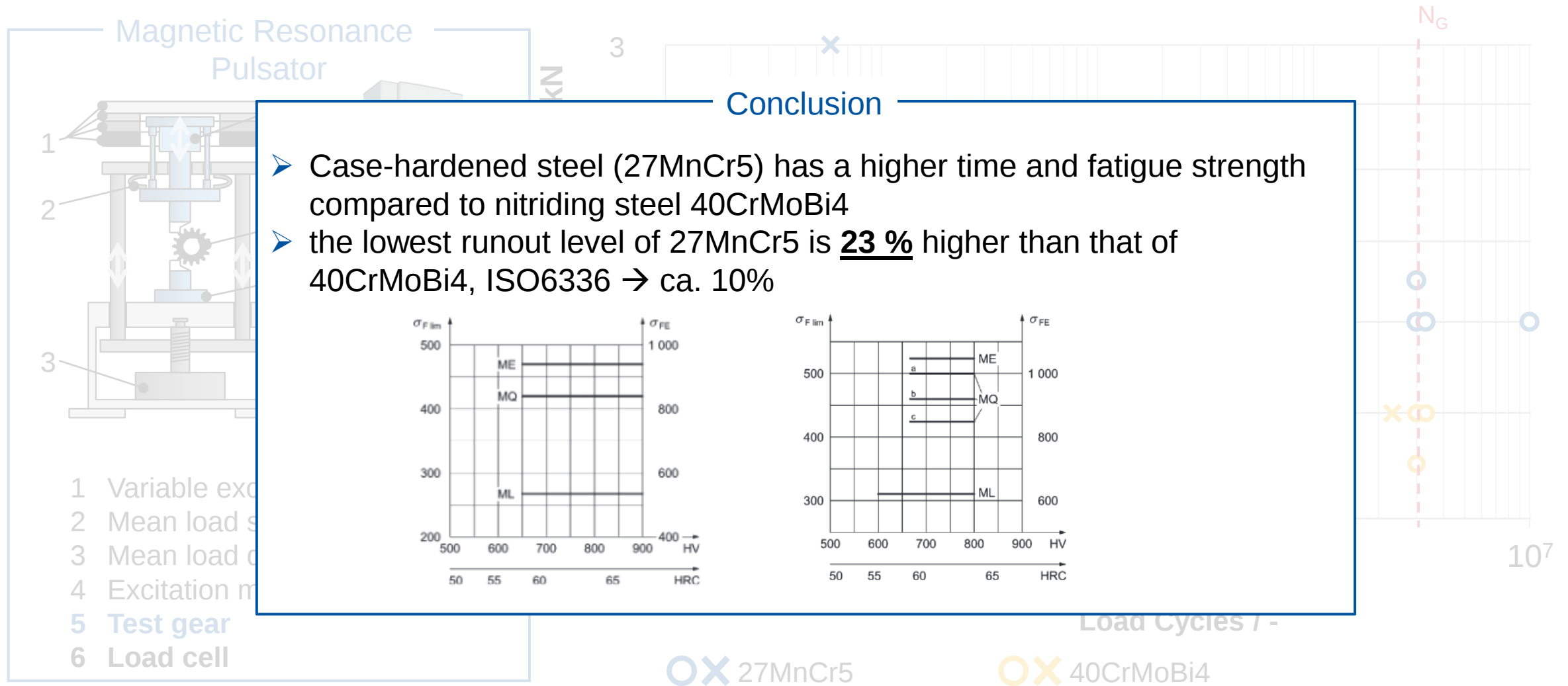
Testing Pulsator Test Rig (Tooth Root Bending)

27MnCr5 Scuffing Geometry / 40CrMoBi4 Scuffing Geometry



Testing Pulsator Test Rig (Tooth Root Bending)

27MnCr5 Scuffing Geometry / 40CrMoBi4 Scuffing Geometry



Agenda

1 Toth Root Bending Strength

2 Flank Load Capacity

3 Scuffing Load Capacity

4 Conclusions

Test rig data

- a = 99.5 mm
- $n_{\max}$ = 12,000 rpm
- $M_{\max}$ = 200 Nm
- Oil = Aeroshell 555
- Q_{oil} = 40 L/min

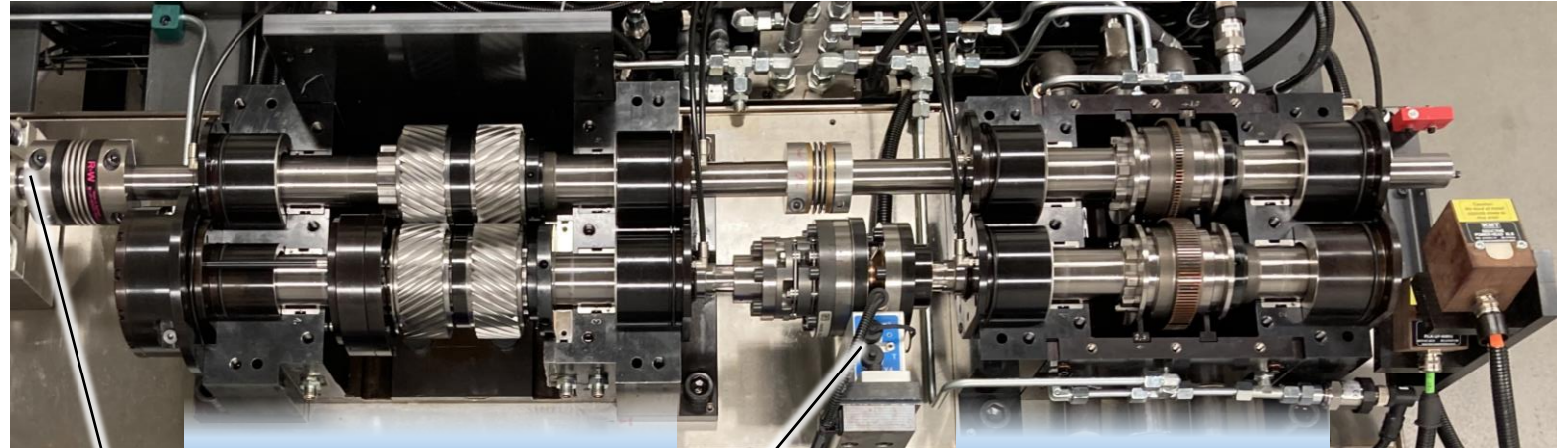


Test geometry
pitting



Test geometry
scuffing

High-speed back-to-back gear test rig



Reference gear box

Test gear box

Shaft from motor

Torque measurement

Predefined parameter:

Centre distance a = 99.5 mm

Tooth ratio z_1 / z_2 = 99/102 or 33/34

Max. torque $M_{\text{in,max}}$ = 200 Nm

Hub – Shaft connection by spline shaft

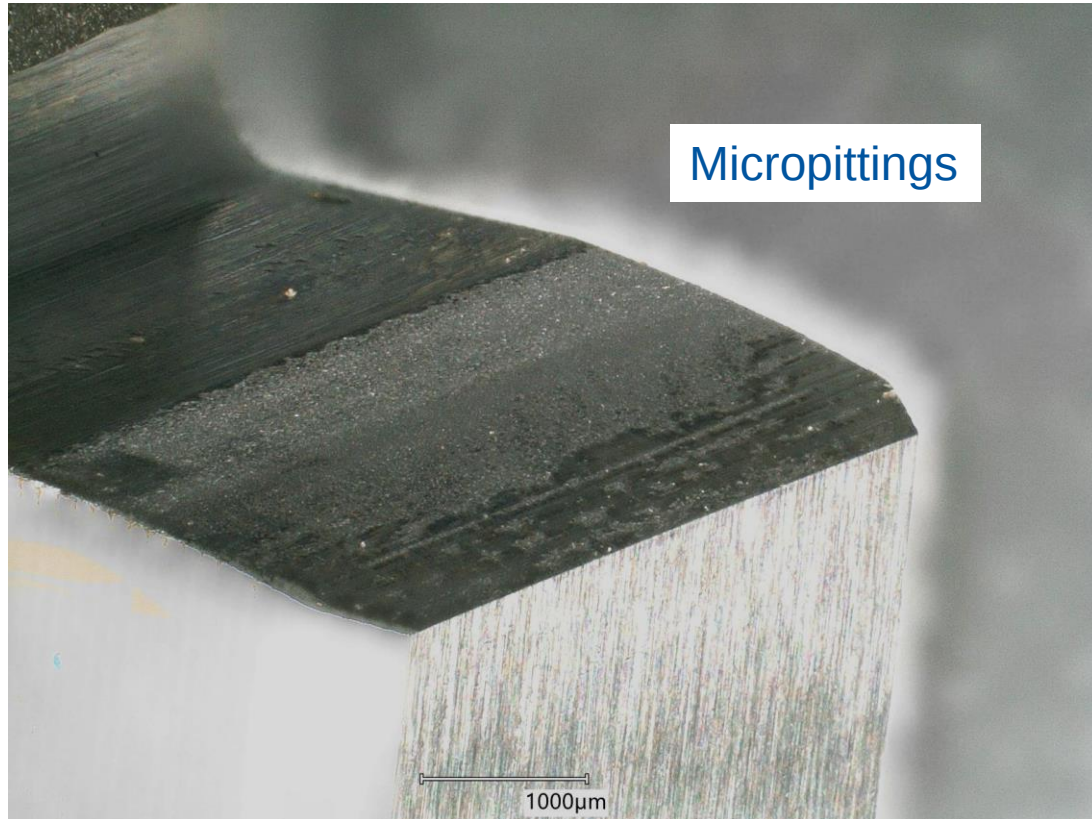
Axial displacement of test wheel due to torque application

Direct comparison of Gars after 50,000,000 Load cycles at 120 Nm and 12,000 RPM

27MnCr5

M = 170 Nm

N = 50,000,000 Load cycles



40CrMoBi4

M = 170 Nm

N = 50,000,000 Load cycles



Testing Mini M4T Test Rig

27MnCr5 Pitting Geometry / 40CrMoBi4 Pitting Geometry

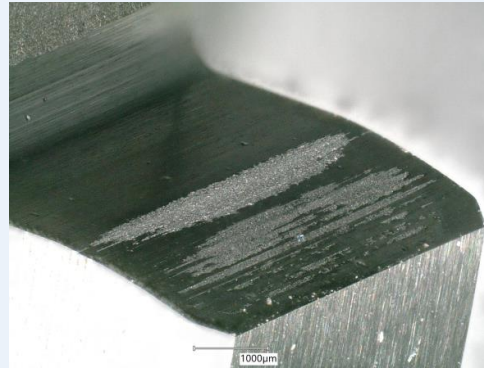
Test gear geometry

- $m_n = 3.06 \text{ mm}$
- $z_1 / z_2 = 33 / 34$
- $b_1 / b_2 = 7 / 25 \text{ mm}$
- $a = 99.5 \text{ mm}$
- $\alpha_n = 18^\circ$
- $\beta = 0^\circ$
- $x_1^* / x_2^* = -0.5073 / -0.3176$
- $d_{a1} / d_{a2} = 101.5 / 103.5 \text{ mm}$

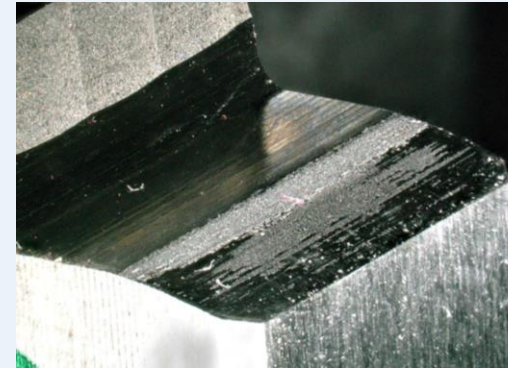
Test data

- $n = 12,000 \text{ rpm}$
- $M = 170 \text{ Nm}$
- $T_{oil} = 40^\circ \text{ C}$
- Oiltype : Aeroshell 555
- Run in: 10 min at 40 Nm

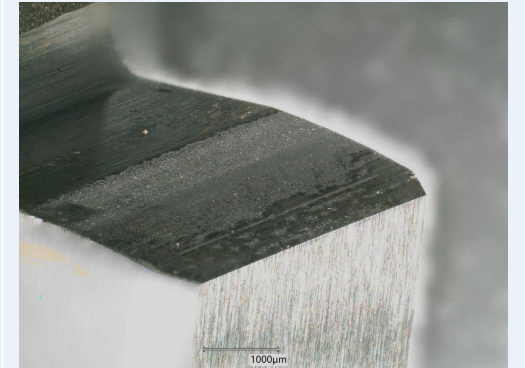
12,000,000 Lastwechsel



24,000,000 Lastwechsel



50,000,000 Lastwechsel

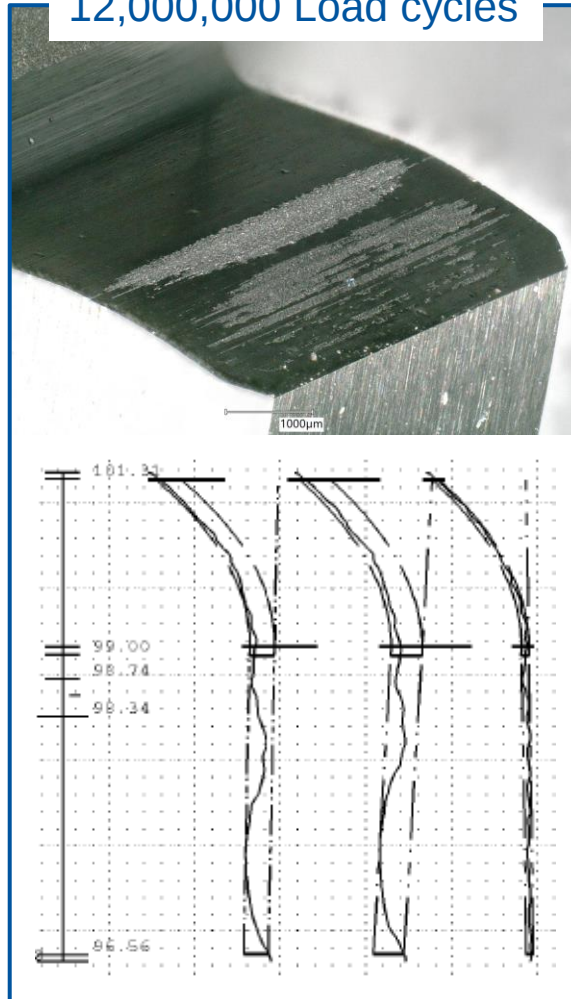


Pinion WSM1-A-28108 Micropitting (170 Nm)

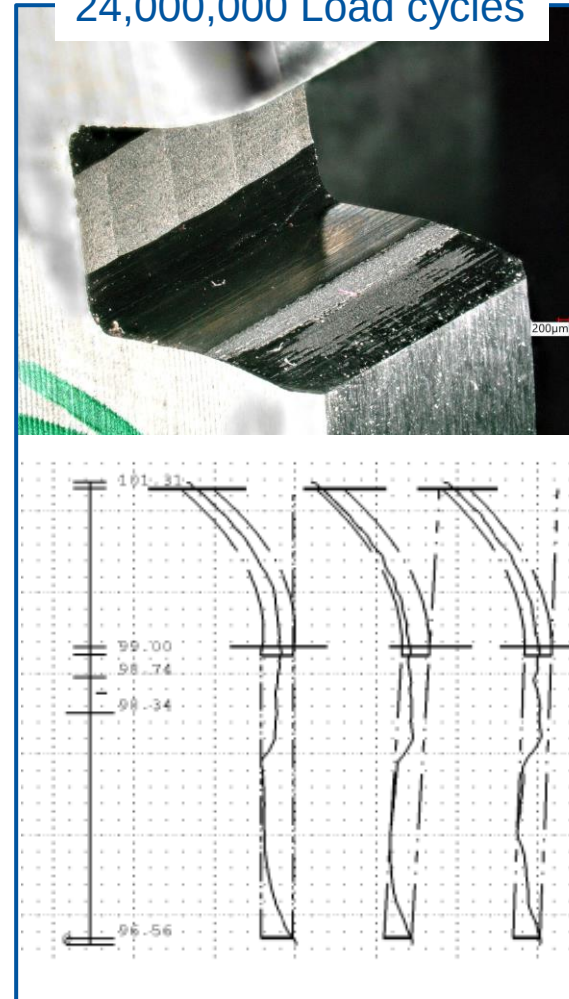
Pitting test

- $m_n = 2.95 \text{ mm}$
- $z_1 / z_2 = 33 / 34$
- $b_1 / b_2 = 7.5 / 25.6 \text{ mm}$
- $a = 99.5 \text{ mm}$
- $\alpha_n = 15^\circ$
- $\beta = 0^\circ$
- $x_1^* / x_2^* = 0.15 / -0.1$

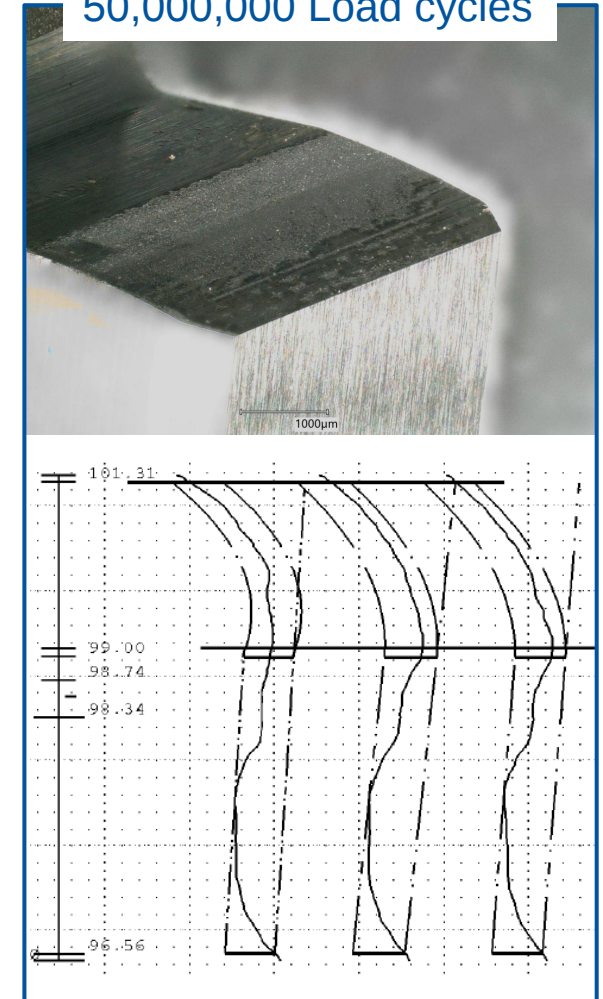
12,000,000 Load cycles



24,000,000 Load cycles



50,000,000 Load cycles

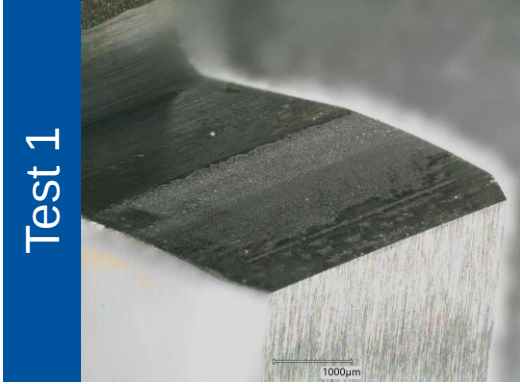


Testing Mini M4T Test Rig

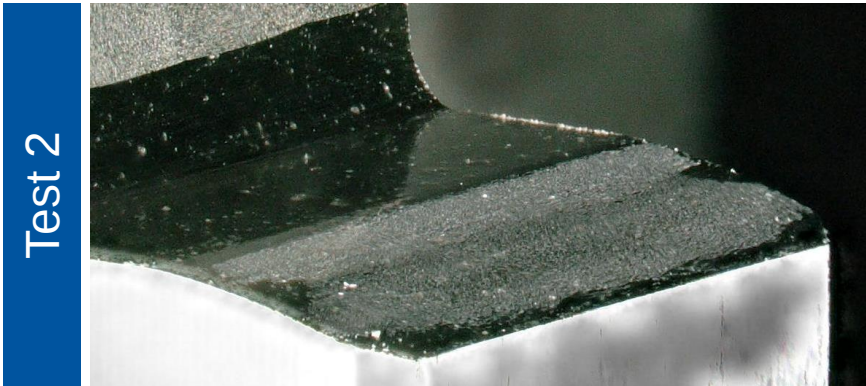
27MnCr5 Pitting Geometry / 40CrMoBi4 Pitting Geometry

27MnCr5 at 170 Nm and 12,000 rpm

Test 1



Test 2



40CrMoBi4 at 170 Nm and 12,000 rpm

Test 1



Test 2



Test 2



Testing Mini M4T Test Rig

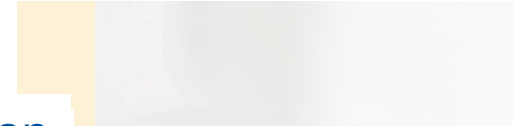
27MnCr5 Pitting Geometry / 40CrMoBi4 Pitting Geometry

27MnCr5 at 170 Nm and 12,000 rpm

Test 1



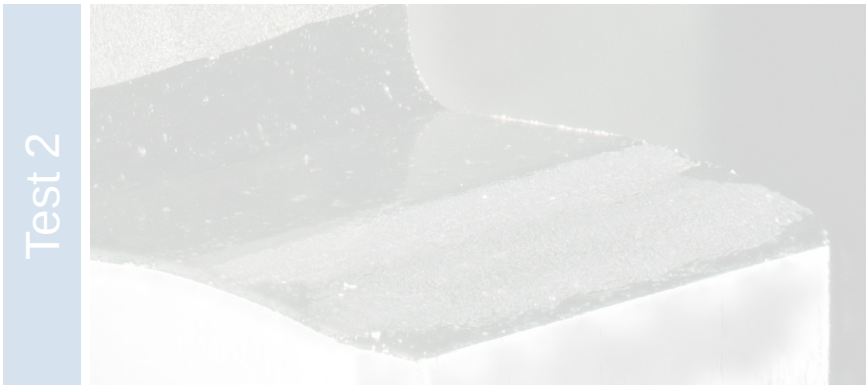
40CrMoBi4 at 170 Nm and 12,000 rpm



Conclusion

- Case-hardened and nitrided steel fail under the same test conditions and with the same surface roughness due to different damage mechanisms
- 27MnCr5 → Micropittings
- 40CrMoBi4 → Pittings / Damage at the tip

Test 2



Test 2



Test 2



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2 Flank Load Capacity

3 Scuffing Load Capacity

4 Conclusions

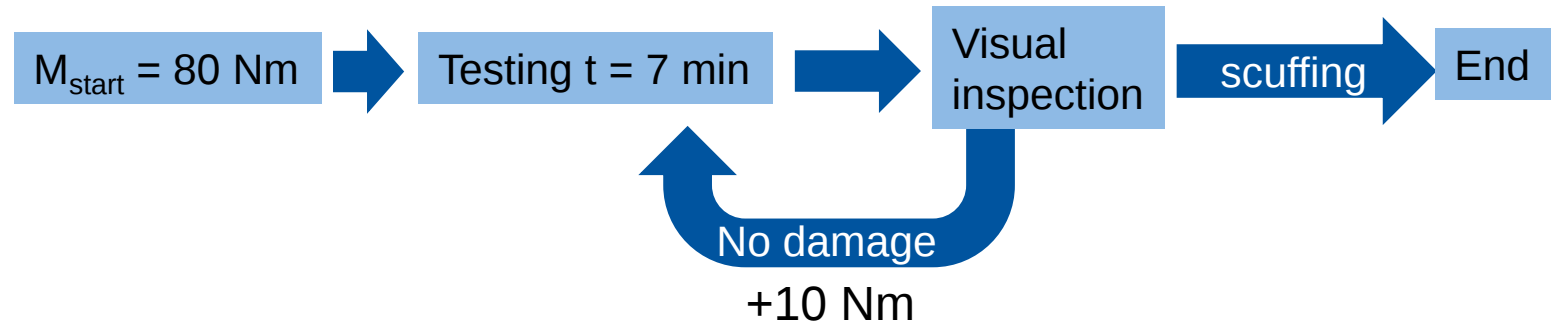
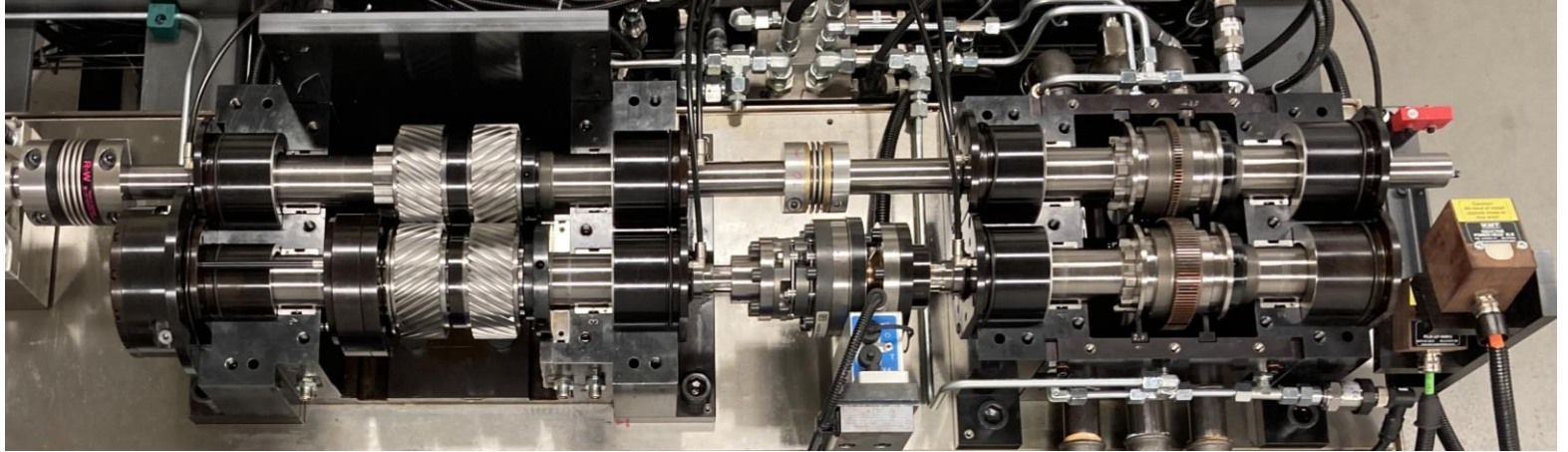
Test rig data

- a = 99.5 mm
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- $M_{\max}$ = 200 Nm
- Oil = Aeroshell 555
- Q_{oil} = 40 L/min



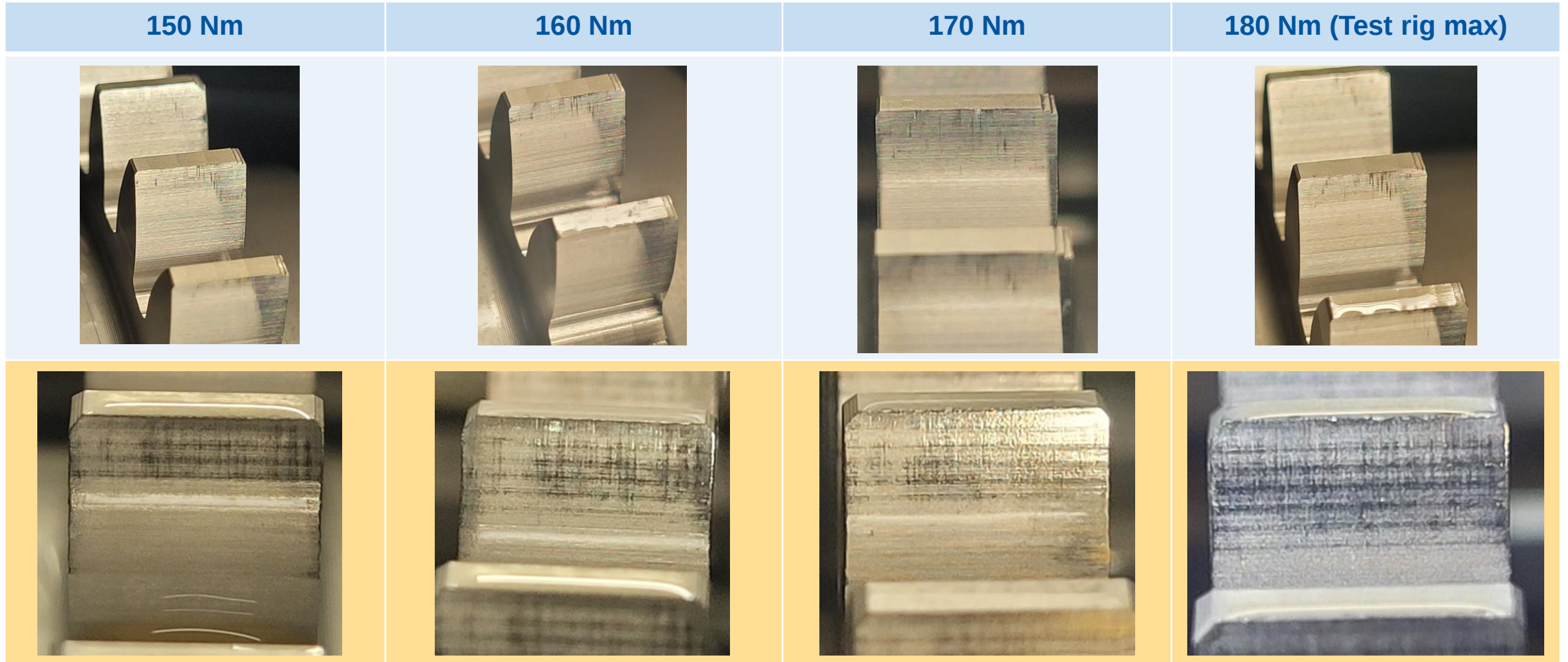
Test geometry
scuffing

High-speed back-to-back gear test rig



Testing Mini M4T Test Rig

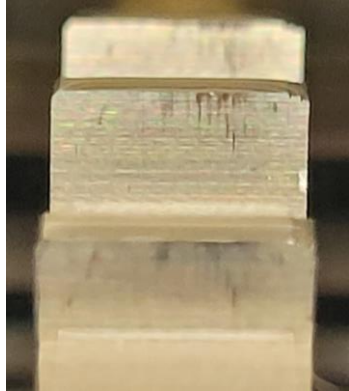
27MnCr5 Scuffing Geometry / 40CrMoBi4 Scuffing Geometry



Testing Mini M4T Test Rig

Scuffing test 27MnCr5 gears at $n = 12000$ rpm

80 Nm



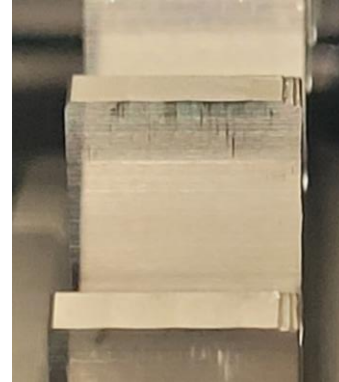
90 Nm



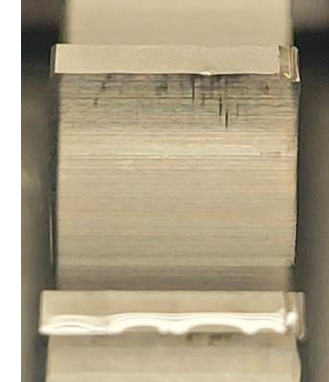
100 Nm



110 Nm



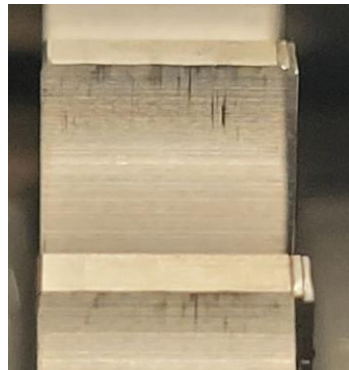
120 Nm



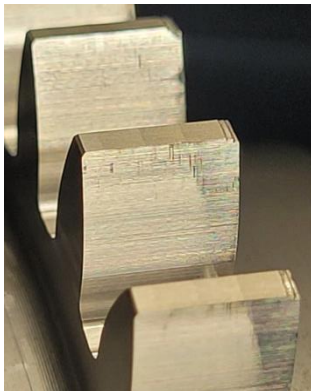
130 Nm



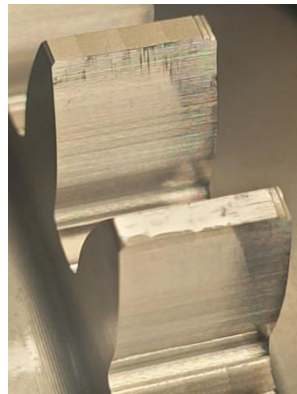
140 Nm



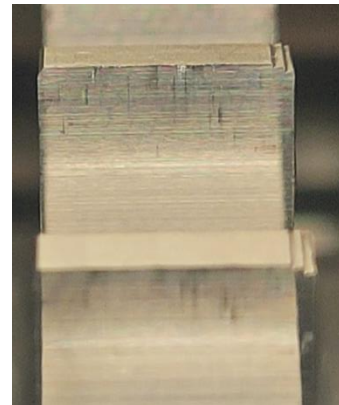
150 Nm



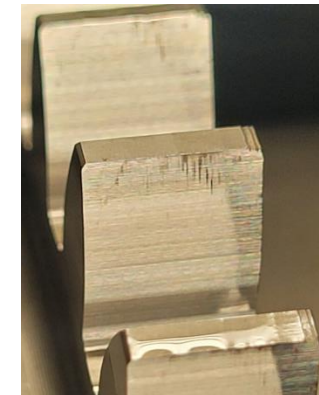
160 Nm



170 Nm



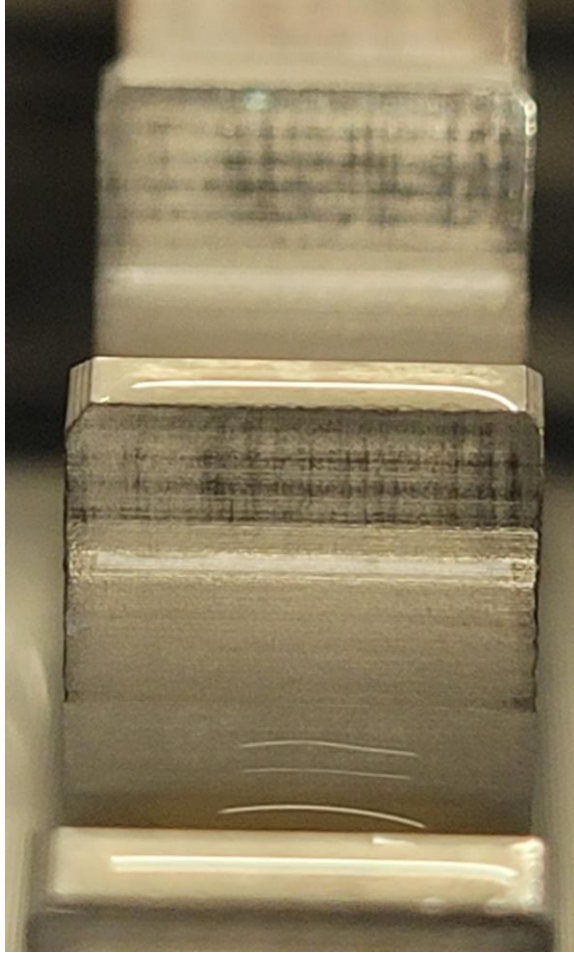
180 Nm



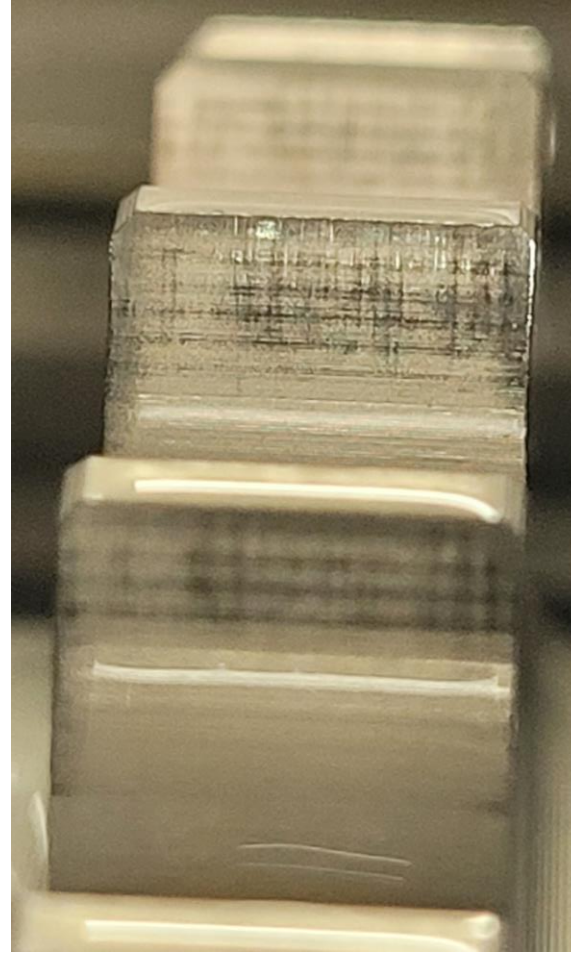
Testing Mini M4T Test Rig

Scuffing test 40CrMoBi4 gears at $n = 12000$ rpm

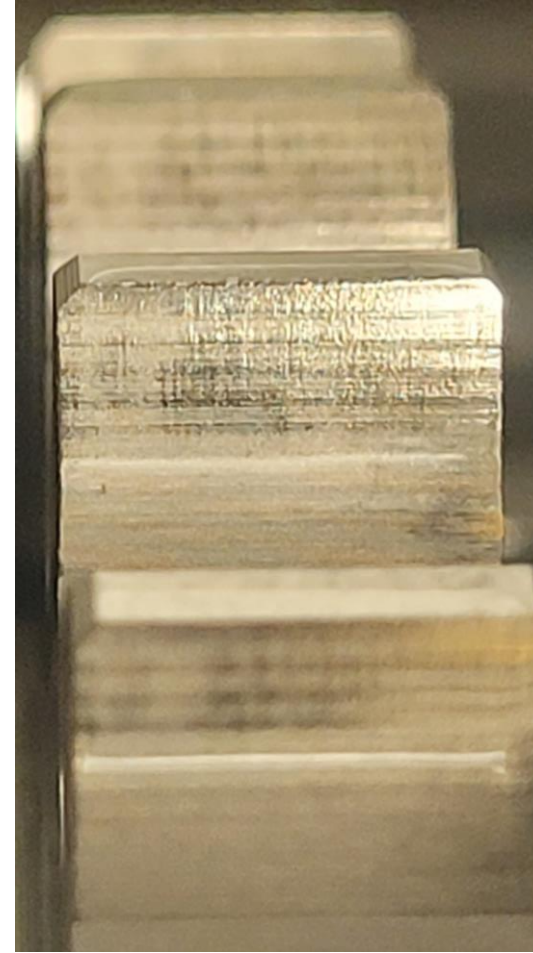
150 Nm



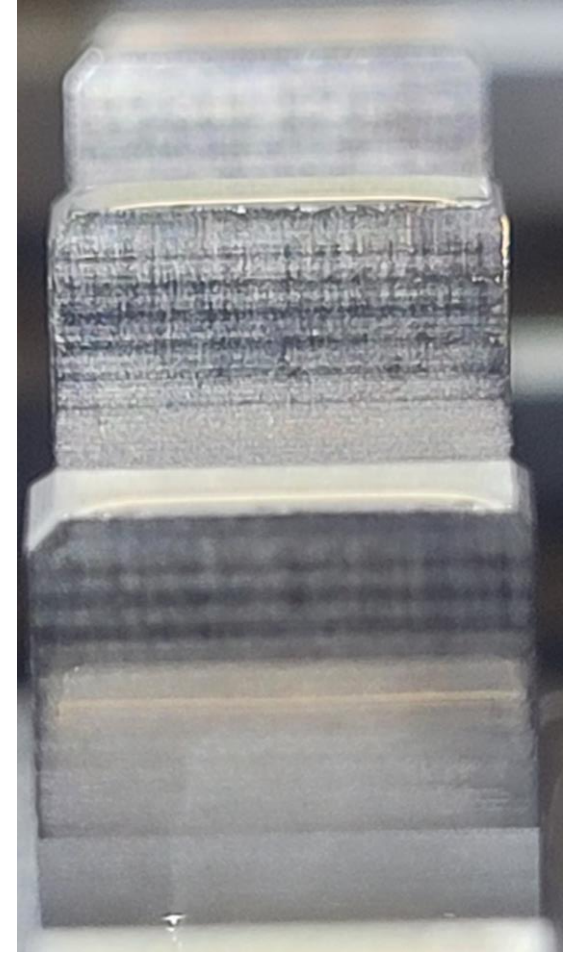
160 Nm



170 Nm



180 Nm



Agenda

1 Toth Root Bending Strength

2 Flank Load Capacity

3 Scuffing Load Capacity

4 Conclusions

Conclusions for Future EV Gears at High Speeds

Project results Topgear and general advices at highspseds

Bending Strength

- 27MnCr5 case-hardened has a higher tooth root bending strength than 40crMoBi4 nitrided steel
- The tooth root load capacity increases at an increased speed of 12000 rpm compared to the standard test at 3000 rpm (ZALFEN)

Flank Load Capacity

- 27MnCr5 case-hardened and 40crMoBi4 nitrided steel show different damage types (Pitting & Micropitting)
- At increased speed, there appears to be an increased risk of micropitting despite good roughness values for case-hardening steel and better lubrication with high tangential velocities in theory
- The initial results show that the pitting load capacity could also increase at higher speeds. This must be investigated further

Scuffing Load Capacity

- The Aerospace 555 oil used does not lead to scuffing regardless of the material variant
- It is known from the state of the art that the scuffing load capacity increases with high circumferential speeds
- Low-viscosity oils with high scuffing load capacity are recommended for high-speed applications